

Innovative Modeling of the Crack Path and Stress Intensity Factor for Arbitrary Shaft Configurations

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Abstract

The lifetime of most engineering structures and components is known to depend on the presence of defects, such as holes, cracks or voids usually introduced during a manufacturing process. In many cases, the crack growth, extension and propagation within a body, still remains a challenging problem in fracture mechanics. The present paper proposes an extended analytical model based on a section method to predict the fracture direction and compute the stress intensity factors for a cracked shaft under mixed-mode loading conditions. The advantage of the present formulation is mainly related to its capability of predicting the direction of crack propagation within a shaft under coupled longitudinal, flexural and torsional loading conditions. The analytical results are straightforwardly compared with the theoretical expressions available from the handbooks and the numerical solutions found with the extended finite element method. The present approach agrees quite well with the theoretical and numerical results already proposed in the literature, thus confirming its potentials for accurate computations of the crack propagation and stress intensity factors for arbitrary configurations.

Keywords

Crack propagation; cracked shafts; extended finite element method; stress intensity factor.

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Introduction

The accurate modeling of 3D cracks in continuum bodies remains a challenging problem in computational mechanics. The importance of computing the fracture parameters and simulating the crack growth in engineering components, stems from the widespread use of numerical fracture mechanics for static and fatigue problems. Fatigue or static failure usually occurs due to the initiation and propagation of single or multiple cracks with different shapes of multiphysical nature along surfaces or nearby them [1, 2]. Despite the large effort by the scientific community for the study of the main factors controlling the crack paths within materials, the results are still not satisfactory. The works by Pook [3, 4] provide an interesting overview

of the active research on several crack path topics of current interest, e.g. fatigue cracking in aircraft structures, fracture toughness testing, fractography, mixed mode fatigue thresholds, crack path stability, failure of laminated and generally jointed structures. A linear elastic study of cracked bodies, crack paths and a collection of some realistic cases can be also found in Pook [5] to study the fatigue crack paths within metallic materials. A large variety of crack path topics, however, makes a useful generalization of the problem within standards and codes quite difficult.

Within a linear elastic fracture mechanics (LEFM) framework, the propagation of a crack is mainly governed by the singular behavior of the stress field in the vicinity of its crack tip. For a 2D quasi-static crack, this behavior is generally governed by

$$\sigma_{ij}(r, \varphi) = \frac{K_I}{\sqrt{2\pi r}} \Sigma_{ij}^I(\varphi) + \frac{K_{II}}{\sqrt{2\pi r}} \Sigma_{ij}^{II}(\varphi) + O(r^0), \quad (1)$$

where $\Sigma_{ij}^I(\varphi)$ and $\Sigma_{ij}^{II}(\varphi)$ are universal functions describing the angular variation of the stress field, and K_I and K_{II} are the stress intensity factors (SIFs). Thus, except for the cases where higher order terms are required to characterize the crack paths, SIFs usually play an important role for a correct characterization of the mechanical properties of cracked materials. The state of art for the stress analysis of cracks, based on the fracture mechanics concept of the SIF, was already reviewed in 1965 by Paris and Sih [6]. Many 2D theoretical solutions of the SIF available from the literature still represent basic formulations for many engineering applications, design procedures, standards and failure assessment codes [7–10].

However, the simplified analyses based on 2D theories, are largely recognized to ignore the 3D effects, including the effects of the plate thickness [11], vertex singularities [12], and coupling of fracture modes I, II and III on the stress and strain fields near the crack front [13–15]. Thus, an accurate computation of the SIFs for 3D surface cracks would be required for practical applications, which is based on the implicit assumption of continuity along a crack front. The understanding of 3D effects has long been recognized as the most important and challenging problem in fracture mechanics, and the increasingly powerful computers have made the numerical investigation possible, especially in the vicinity of corner points. In the last decades, a large amount of theoretical and experimental information about the 3D effects has been discussed by several researchers, as it can be seen in the overview provided by Pook [16], or other recent works from the literature [17–21].

There is an important class of crack problems, which involve the surface damage in the form of part-through cracks in round bars or beams, where a 3D analysis is involved. Miao et al. [22] analyzed numerically 3D mixed mode central cracked plates under biaxial loading conditions, by including the elastic and elastic-plastic J -integrals. Valiente [23] obtained numerically the solutions for the sample compliance and the energy release rate based on the finite element approach. The stiffness derivative technique was applied by the same author to determine the energy release rate, based on the virtual crack extension. Levan and Royer [24] and Couroneau and Royer [25] applied the boundary integral equation method to determine the SIFs for round bars with transverse circular cracks. The finite element approach was then applied by Daoud and Cartwright [26, 27] to compute the SIFs for a circular bar with a straight crack under tension and bending. Ng and Fenner [28]

built a 3D finite element model to treat fracture mechanics and to determine the SIF for a cracked circular bar with a straight and circular crack fronts on one side, when subjected to an axial and flexural loading.

Shin and Cai [29] compared some experimental and FEM-based results about the SIFs for an elliptical crack within a circular shaft under tension and bending. Carpinteri [30] computed numerically the dimensionless SIFs for a straight-front and semi-elliptical crack by using a 3D finite element modeling. Erjani [31] described the normalized SIFs as a function of the crack depth for round bars under axial forces and bending moments. Naik and Maiti [32] presented the formulation for analyzing the triply coupled free vibration of beams based on a compliance approach in the open edge crack in an arbitrary angular orientation. Some reviews of the SIF solutions for surface cracks in round bars subjected to tension loading can be found in the literature [33, 34]. Due to the complexity of cracked problems, a large variety of numerical techniques have been employed in the literature to determine 3D SIFs nearby a crack. In this context, the finite element method (FEM) is the most commonly used procedure for computing the SIFs due to the progressive increase in the computational power and FEM-based codes available for computational studies. Since 1969, Dixon and Pook [35] announced the adoption of the FEM as a useful tool to predict the SIFs in the study of opening mode crack growth problems where the nominal stress level is known to be lower than the yield stress of a material. Chen and Wang [36] determined the mixed mode geometry factors for a finite plate with an inclined edge crack by means of a weight function approach. This was also adopted by the same authors to calculate the mixed-mode SIFs and T-stress for an offset double edge-cracked plate. Jogdand and Murthy [37] proposed an innovative method for estimating the mixed-mode SIFs based on the formation of over-determined system of equations, which was then implemented in existing finite element codes. A FEM-based dislocation model was also proposed by Feng et al. [38] in order to determine the SIFs for an arbitrary number of through-thickness cracks. Whitcomb et al. [39] and Chaudhuri [40] commented on the accuracy of FEM results, based on a displacement potential energy approach, in the neighborhood of stress discontinuities and stress singularities. As mentioned in [39, 40], the numerical distribution of the stress and/or displacement as obtained with FEM for cracked specimens usually loses the accuracy in the closest region around the discontinuity or singularity.

The SIFs determined from local displacements or stresses, require some unconventional crack tip elements, also known as quarter point elements, which

account for the $1/\sqrt{r}$ singular stress behavior of the crack tip. A quarter-point element is obtained from the standard quadratic element by simply moving the mid-node coordinates three-fourths of the side towards the crack tip. It is well known from the literature that the solution can converge quite slowly for cracked problems, when adopting the only conventional elements, which do not include stress singularities. It seems that the most common approach for modeling 3D cracks using the FEM is to introduce the singularity stress behavior in the crack tip elements by relocating the element mid-side nodes to new locations. This introduces a singularity in the Jacobian inverse of the geometric transformation. Closed-form solutions for the SIFs are available for simple crack geometries in 3D. However, if the crack surface is not aligned with the element boundaries, the displacement discontinuity and the traction conditions on the crack surface cannot be treated by standard FEMs. In the actual cracked structures, the numerical prediction of the fracturing process is very hard to perform, especially for arbitrarily-shaped cracks in finite specimens. Based on the above mentioned numerical limitations of FEM when treating physical models in presence of cracks, some alternative numerical strategies have been recently developed by the scientific community to analyze the fracture process of elastic structures. To this end, the Cell Method [41, 42] the Embedded Mesh formulation [43, 44], the Generalized Differential Quadrature (GDQ) Method [45–48], the Isogeometric Method [49–52], or the extended FEM (XFEM) [53–55] have been proposed as computationally efficient techniques to treat the problem. In this study, we have employed the XFEM to treat stress singularities in fracture mechanics for comparative purposes, as introduced by Belytschko and Black [56]. This approach enables a local enrichment of the discrete space, based on the concept of partition of unity. This results to be very useful for the approximation of solutions with a pronounced non-smooth behavior in some parts of the computational domain (i.e. near discontinuities or singularities), where a standard FEM could be insufficient to offer good quality results. The XFEM offers significant advantages by enabling optimal convergence for these applications, since it alleviates the main shortcomings of a classical FEM approach associated with meshing of the cracked surface. The FEM would represent the essence of the XFEM, and hence a lot of the theoretical and numerical developments in finite elements can be readily extended and applied. The XFEM-based approach splits a modeling problem in two distinct parts: the mesh generation for the geometric domain (without considering any possible crack), and the enrichment step of the discrete

approximation by means of additional functions that model a geometric discontinuity. Anyway, the fracture criterion is the primary basis for the safe evaluation in cracked materials. A theoretically integrated fracture criterion should be capable to predict the fracture direction and loading of cracked structures under varying loading conditions.

In this sense, different fracture criteria were developed in the past decades to describe a mixed-mode fracturing. Erdogan and Sih [57] proposed the maximum circumferential stress criterion, which assumed that fracture occurs in the direction where the circumferential stress surrounding the crack tip reaches the maximum value. Smith et al. [58] developed a generalized maximum tensile stress criterion based on the T-stress. A generalized maximum tangential stress criterion was employed in Ayatollahi et al. [59–61] to predict the fracture initiation angles, which is strongly dependent on the elastic T-stress, under mixed mode loading conditions in different brittle or composite materials. Sih [62, 63] introduced the minimum strain energy density factor criterion which assumes that fracture occurs in the direction where the strain energy density reaches the minimum value. A similar fracture criterion was also proposed by Richard et al. [64] to treat 3D-mixed mode problems, which revealed to be suitable for the description of the crack growth, in agreement with [62].

Additional bases and applications of the averaged strain energy density criterion for the computation of notch SIFs for coarse meshes, are detailed in Lazzarin et al. [65, 66]. The T-stress effects in the application of a conventional minimum strain energy density criterion has been also recently discussed by Ayatollahi et al. [67], by considering higher order terms in the William's series expansion. A clear influence of the magnitude and sign of the T-stress on the crack trajectory and the fracture strength of different specimens is highlighted by the authors in their work. Hussain et al. [68] proposed the maximum energy release rate criterion based on Griffith's theory [69]. Cornetti et al. [70] introduced a fracture criterion based on a double enforcement of the stress and energy balance, named as finite fracture mechanics, to describe the finite crack propagation and advancement, recently applied for mode-I loaded specimens in Dimitri et al. [71].

A criterion for 3D cracking under mixed-mode loading conditions, was additionally described by Schollmann et al. [72]. The criterion allows for the prediction of the 3D crack advancement with the help of two different deflection angles. Similarly, Pook [73] proposed a 3D criterion based on the definition of two crack deflection angles. Chang et al. [74] presented

a general mixed-mode brittle fracture criterion based on the concept of the maximum potential energy release rate. Additional criteria of fracturing are reviewed and discussed in [75, 76] together with their advantages and drawbacks for different applications. For an engineering estimation of the maximum strength for cracked bodies, the use of efficient procedures involving a limited computational effort can be suitable, also for lower levels of accuracy. The present paper introduces a simple engineering method to calculate the SIF, based on the section method, as introduced in [77] and modified by Nobile [78, 79]. The section method considers the stress singularity at the tip of an elastic crack, and is based on an elementary beam theory equilibrium condition for internal forces evaluated in the cross-section passing through the crack tip. The derived simple formulation for SIFs of transversely cracked circular section beams subjected to bending and torsional moment, as well as to the shear force, are in a reasonable agreement with the results from the available literature. Moreover, three different criteria as proposed by Richard [64], by Sih [62, 63], and by Schollmann et al. [72] are used here to determine the crack direction of propagation. The results, both in terms of SIFs and crack angles of propagation, are compared with the available results from the literature as well as with results obtained with the XFEM as implemented in the ABAQUS code.

Analytical Modeling

Problem Statement. Let us consider a 3D structure under mixed-mode loading condition, which is made of two perpendicular circular beams subjected to a vertical concentration load F at the free ending. The lengths of the two beams are L and $L/2$, respectively, while the crack is located in the middle of the first beam of length L . The structure has a circular section with radius R and a crack length a (see Fig. 1). The cracked section is subjected to a shear force $V = F$, a bending moment $M = FL/2$, and a torsional moment $T = FL/2$. For this structure, we have implemented the section method as proposed

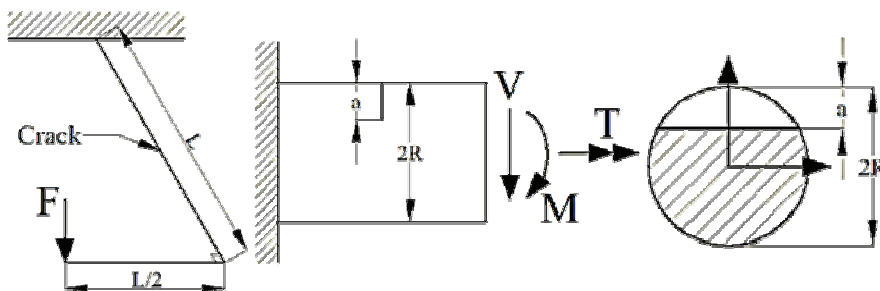


Fig. 1. The structural problem

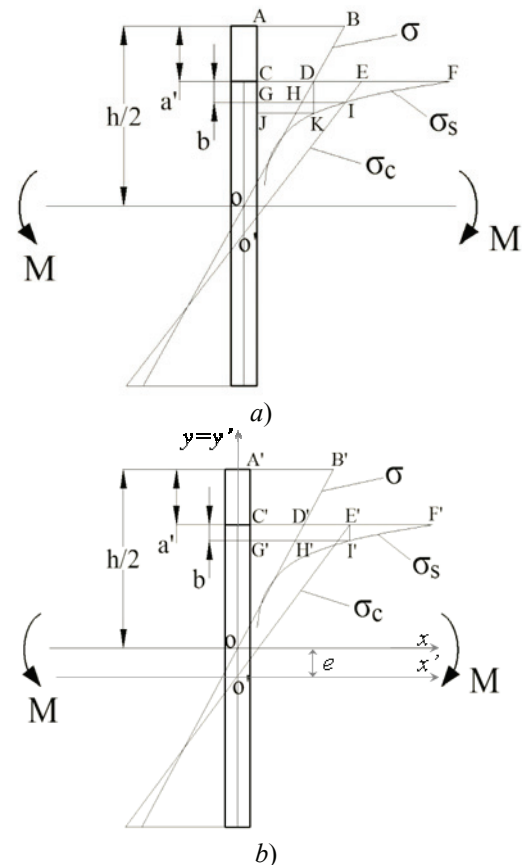


Fig. 2. Rectangular cross section and stress distribution

by Parton and Morozov [77] in order to compute the SIF. Based on this method, the crack tip is subjected to an additional force related to an increase in the stress expressed as

$$\Delta\sigma = \int_0^b \sigma_s dr, \quad (2)$$

where b is the length, σ_s is the nominal stress, and $\Delta\sigma$ is an additional force per unit length, yielding to the equivalence between the areas A_{ABDC} and A_{CFIKJ} (see Fig. 2, a).

Nobile [78, 79] introduced a slight variation in the section method, i.e. the additional stress $\Delta\sigma$ due to the cracked section was computed through a balance between the equivalent force of the areas A_{CFIG} and A_{CEIG} (see Fig. 2, a). In other words, b is the distance where the singular stress is equal to the normal stress when the neutral axis moves along CG (see Fig. 2, a). As clearly seen from Fig. 2, a, however, the force associated with the area A_{CEIG} is lower than the force associated with the area A_{CFIG} . This would make the assumption by [78, 79] almost inaccurate. Thus, in this work we propose an improved

version of the section method, starting from the original section method by Parton and Morozov [77] and modified by Nobile [78, 79]. Based on our improved version, the additional force due to the stress concentration at the crack tip is computed by comparing the equivalent forces associated to the areas $A_{C'D'E'F'T'H'G'}$ and $A_{A'B'D'H'G'C'}$ in Fig. 2, *b*. The distance b can be determined from the following balance condition (see Fig. 2, *b*)

$$\sigma_s|_{r=b} = \sigma_c|_{y'=\bar{y}}, \quad (3)$$

where \bar{y} is the distance of the reduced cross section at the crack tip from the neutral axis.

Rectangular Beam under a Bending Moment. As reference example, we start the work considering a beam with a rectangular cross section of thickness $t = 1$, height h , and a straight through transverse crack of length a . The cross section is subjected to a bending moment M , as seen in Fig. 2, *b*, with a linear distribution of the normal stress expressed as

$$\sigma = \frac{M}{I} y, \quad (4)$$

where $I = th^3/12$ is the second moment of inertia for the uncracked cross section.

Similarly, the reduced portion of the cross section due to the presence of the crack, features the following linear distribution

$$\sigma_c = \frac{M}{I_c} y', \quad (5)$$

where $I_c = t(h-a')^3/12$ is the corresponding second moment of inertia, $y' = y + e$ is the general distance of a point in the system of coordinates $O'x'y'$, $e = OO' = a'/2$ is the distance of the neutral axis from the x -axis. It is well known from the literature that the stress component σ_s in mode-I condition near a singular corner point or discontinuity, is related to the SIF, defined as

$$\sigma_s = \frac{K_I}{\sqrt{2\pi r}}, \quad (6)$$

where r is the general distance from the crack tip.

The equilibrium of the forces in the horizontal direction reads

$$\int_{\frac{h}{2}-a'-b}^{\frac{h}{2}} \sigma dy = \int_0^b \sigma_s dr, \quad (7)$$

where the exact value of the distance b can be found by comparing Eqs. (5) and (6), i.e.

$$\frac{K_I}{\sqrt{2\pi b}} = \frac{M}{I_c} y. \quad (8)$$

After some algebraic manipulations, the mode-I SIF can be expressed as follows

$$K_I = \frac{6M}{th^{3/2}} F(\eta), \quad (9)$$

where $\eta = a'/h$ is the dimensionless crack depth, whereas the geometry function is defined as

$$F(\eta) = \frac{\sqrt{2\pi\beta}}{(1-\eta)^2}, \quad (10)$$

with

$$\beta = \begin{cases} \frac{1}{2} - \eta - \frac{1}{(1-\eta)^2} + \sqrt{\frac{1}{(1-\eta)^4} + \frac{2\eta-1}{(1-\eta)^2} + \frac{1}{4}} & \text{for } 0.5 - \eta - \beta \geq 0 \\ \frac{1}{2} - \eta + \frac{1}{(1-\eta)^2} - \sqrt{\frac{1}{(1-\eta)^4} - \frac{2\eta-1}{(1-\eta)^2} + \frac{1}{4}} & \text{elsewhere.} \end{cases} \quad (11)$$

The geometry function $F(\eta)$ defined by the Eqs. (10) and (11) can be compared with the corresponding expressions given by Brown [80], Tada [8] and Nobile [78]:

– least square fitting method, precision of 0.2 % for $\eta \leq 0.6$ [80]

$$F(\eta) = \sqrt{\pi\eta} (1.122 - 1.40\eta + 7.33\eta^2 - 13.08\eta^3 + 14.0\eta^4); \quad (12)$$

– precision better than 0.5 % for any η [8]

$$F(\eta) = \sqrt{2 \tan \frac{\pi\eta}{2}} \frac{0.923 + 0.199 \left(1 - \sin \frac{\pi\eta}{2}\right)^4}{\cos \frac{\pi\eta}{2}}; \quad (13)$$

– modified section method [78]

$$F(\eta) = \frac{0.482}{\sqrt{(1-\eta)^3}}. \quad (14)$$

The main analytical results in terms of geometry functions are plotted comparatively in Fig. 3, where it is worth noticing the good agreement between our

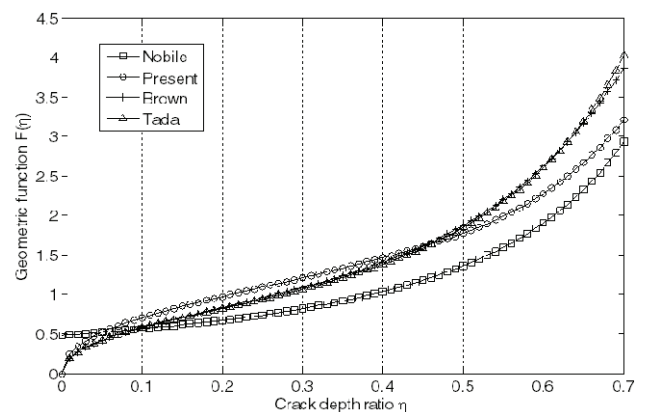


Fig. 3. Geometric function for rectangular beam

proposed formulation and the results by Brown [80], Tada [8] and Nobile [78], especially for $\eta \leq 0.6$. A relative difference of few units percent can be generally noticed with respect to the predictions by Brown [80] or Tada [8], with larger differences with respect to the predictions by Nobile [78].

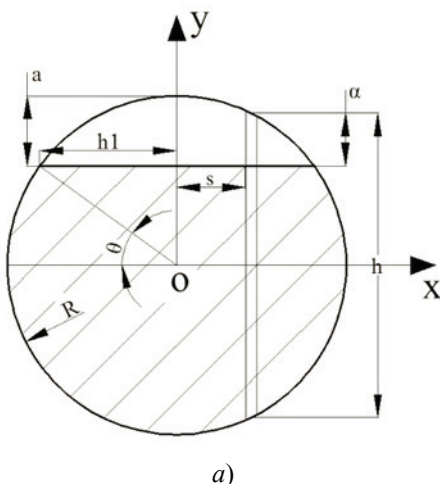
Circular Beam. The previous analysis is now repeated for a circular beam with radius R and precrack length a (see Fig. 4, a) subjected to a bending moment, shear force, and torsional moment, alternatively. Due to the symmetry of the problem, we analyze only half of the geometry in what follows.

Bending Moment. The cracked circular section is here assumed to be divided into independent rectangular strips of width t , which do not exchange any internal stress along their interface. This approximation would be less appropriate near the ends of the crack tip. Each strip has a general height h and crack depth a' which depend on the general distance s of the strip from the y -axis, as follows:

$$\begin{aligned} h &= 2\sqrt{R^2 - s^2}; \\ a' &= \alpha = \frac{h}{2} - (R - a). \end{aligned} \quad (15)$$

The distribution of the normal stresses for the uncracked cross section is defined by Eq. (4), where I is the second moment of inertia for the uncracked circular section. Similarly, the distribution of the normal stresses for the reduced cross section passing through the crack tip is defined by Eq. (5), where I_c is the second moment of inertia for the reduced portion of the circular cross section.

The singular stress component is related to the mode-I SIF as in the Eq. (6), whereas the equilibrium of the forces along the longitudinal direction of the beam (z -axis) is expressed as



a)

$$\int_{\frac{h}{2}-a-b}^{h/2} \sigma dy = \int_0^b \sigma_s dr. \quad (16)$$

The value of b in the Eq. (16) is found from the equilibrium

$$\frac{K_I}{\sqrt{2\pi b}} = \frac{M}{I_c} y' \Big|_{y'=y=h/2-\alpha+e}, \quad (17)$$

while the combination of Eqs. (16) and (17) yields to the following expression for the SIF

$$K_I = \frac{M}{R^{2.5}} F(\eta), \quad (18)$$

where $\eta = \alpha/h$ is the crack depth ratio, and $F(\eta)$ is defined as

$$F(\eta) = \frac{R^4 \sqrt{2\pi\beta}}{2I_c} \left(\frac{R}{h}\right)^{3/2} (1-\eta), \quad (19)$$

with

$$\eta = \frac{1}{2} + \left(2\frac{a}{D} - 1\right) \frac{1}{2\sqrt{1 - \left(\frac{s}{h_1}\right)^2 \left(1 - \left(1 - 2\frac{a}{D}\right)^2\right)}} \quad (20)$$

and

$$\frac{R}{h} = \frac{\left(2\frac{s}{h_1} - 1\right)}{2\left(2\frac{a}{D} - 1\right)}. \quad (21)$$

The geometry function $F(\eta)$ is plotted in Fig. 4, b vs. the dimensionless ratio s/h_1 for different lengths of the precrack a/D . As predicted in the literature [8], the value of $F(\eta)$ decreases for increasing distances from the y -axis and decreasing values of the dimensionless ratio a/D . The analytical results based on our proposed formulation agree quite well with

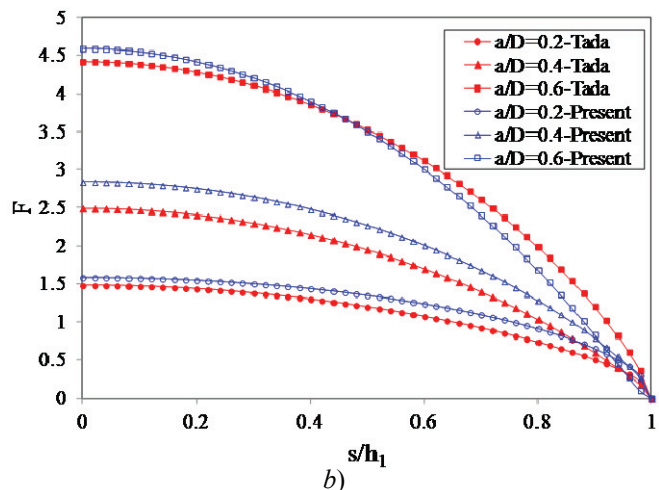


Fig. 4. Circular cross section (a) and geometry function vs. the dimensionless ratio s/h_1 for K_I (b)

the predictions by Tada [8] adapted for a circular section, given by

$$F(\eta)_{\text{Tada}} = \frac{4}{\pi} \sqrt{\frac{h}{R} \left(1 - \left(\frac{s}{R}\right)^2\right)} \sqrt{2 \tan \frac{\pi \eta}{2} \frac{0.923 + 0.199 \left(1 - \sin \frac{\pi \eta}{2}\right)^4}{\cos \frac{\pi \eta}{2}}} \quad (22)$$

For K_I , the relative difference between results seems to increase for higher values of the ratios a/D and s/h_1 .

Shear Force. Consider now the circular cross section in Fig. 5, *a* subjected to a shear force V . In this case, the actual direction of the transverse shear stress would depend on the general distance $(R-a)$, such that we could no longer assume that the shear stress acts parallel to the y -axis. Considering, first, the only contribution of the shear stress along the y -direction, we have

$$\tau_{zy} = \frac{4V \cos \vartheta}{3\pi R^2} = \frac{4V(R^2 - y^2)}{3\pi R^4}, \quad (23)$$

where $\cos \vartheta = \sqrt{2Ra - a^2}/R$ and $h_1 = \sqrt{R^2 - y^2}$.

For $y=0$, the vertical shear stress τ_{zy} reaches the maximum value equal to

$$\tau_{zy_max} = \frac{4V}{3\pi R^2}. \quad (24)$$

The vertical shear stress contributes to the mode-II fracturing. As described in [79], we approximate the average shear stress acting on the reduced cross-section throughout the crack tip with the average shear stress acting on the un-cracked cross section. Thus, we apply the same method by Parton and Morozov [77] to compute the SIF.

The singular stress due to crack is related to the mode-II SIF as follows

$$\tau_{zy_s} = \frac{K_{II_V}}{\sqrt{2\pi r}}. \quad (25)$$

The equilibrium of the forces along the longitudinal direction of the beam can be written as

$$\int_{\frac{h}{2}-\alpha}^{\frac{h}{2}} \tau_{zy} dy = \int_0^b \tau_{zy_s} dr, \quad (26)$$

where b can be determined by enforcing the following condition

$$\tau_{zy_s} |_{r=b} = \tau_{zy} |_{y=\frac{h}{2}-\alpha}. \quad (27)$$

Upon substitution of Eqs. (23) and (25) into Eqs. (26) and (27), it is

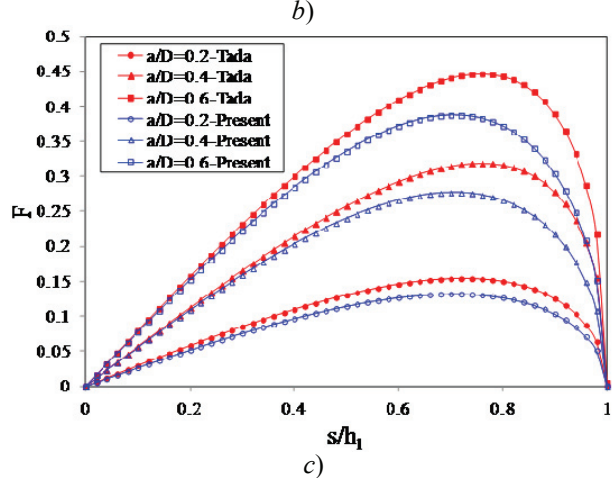
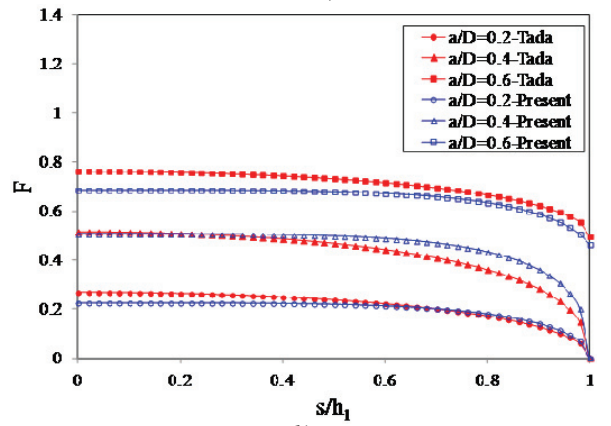
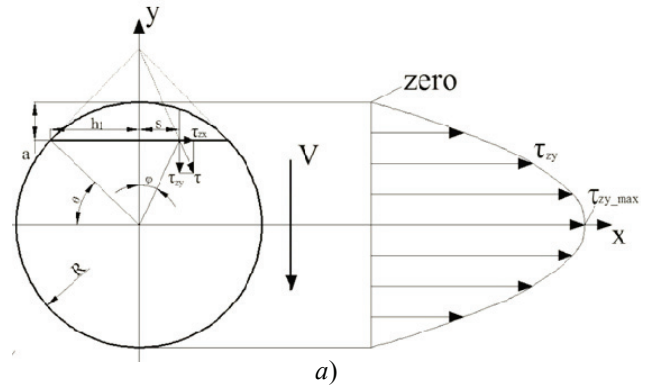


Fig. 5. Approximated shear stress distribution within the circular cross section (*a*), and geometry functions for K_{II_V} (*b*) K_{III_V} (*c*) due to the shear stress vs. the dimensionless ratio s/h_1

$$K_{II_V} = \sqrt{2\pi\beta} \frac{4V}{3\pi R^{1.5}} \sqrt{\frac{h}{R} \left(1 - \left(\frac{h}{R}\right)^2 (1-\eta)^2\right)}, \quad (28)$$

with

$$\beta = \frac{b}{h} = \frac{\left(-\frac{1}{3}\eta^3 + \frac{1}{2}\eta^2 + \left(\frac{R}{h}\right)^2 \eta - \frac{1}{4}\right)}{2\left(\left(\frac{R}{h}\right)^2 - \left(\frac{1}{2} - \eta\right)^2\right)}. \quad (29)$$

By substituting Eq. (29) into Eq. (28), the mode-II SIF reads

$$K_{II-V} = \frac{V}{R^{1.5}} F(\eta), \quad (30)$$

where the geometry function is

$$F(\eta) = \frac{4}{3\pi} \sqrt{2\pi\beta} \sqrt{\frac{h}{R}} \left(1 - \left(\frac{h}{R} \right)^2 (1-\eta)^2 \right). \quad (31)$$

Figure 5, *b* plots the geometry function $F(\eta)$ vs. the dimensionless ratio s/h_1 for different lengths of the precrack a/D . A comparative assessment of the proposed formulation is performed with respect to previsions by Tada [8], adapted for a circular section, given by

$$F(\eta)_{\text{Tada}} = \frac{1}{\pi} \sqrt{\frac{h}{R}} \pi \eta \frac{1.3 - 0.65\eta + 0.37\eta^2 + 0.28\eta^3}{\sqrt{1-\eta}} \quad (32)$$

where a good matching between the two approaches can be noticed especially for lower values of a/D . An increasing difference between the proposed approach and predictions by Tada [8], instead, can be observed for higher values of a/D , whereas $F(\eta)$ never gets a null value in the present formulation even when $s/h_1 = 1$.

Considering now, the additional shear stress component τ_{zx} along the x -direction, expressed as

$$\tau_{zx} = \frac{s}{h_1} \tau_{zy} \tan \theta = \frac{4sV \sin \theta}{3\pi R^3} = \frac{4Vs_y}{3\pi R^4} \quad (33)$$

whose singular value is related to the mode-III SIF as

$$\tau_{zx-s} = \frac{K_{III-V}}{\sqrt{2\pi r}} \quad (34)$$

the equilibrium of the forces along the x -direction of the beam is expressed as

$$\int_{\frac{h}{2}-\alpha}^{h/2} \tau_{zx} dy = \int_0^b \tau_{zx-s} dr, \quad (35)$$

where b can be determined by enforcing the following condition

$$\tau_{zx-s} \big|_{r=b} = \tau_{zx} \big|_{y=\frac{h}{2}-\alpha}. \quad (36)$$

The substitution of Eqs. (33) and (34) into Eqs. (35) and (36) gives

$$K_{III-V} = \frac{4Vs}{3\pi R^{2.5}} \left(\frac{h}{R} \right)^{1.5} \left(\frac{1}{2} - \eta \right) \sqrt{2\pi\beta} \quad (37)$$

and

$$\beta = \frac{(1-\eta)\eta}{2(1-2\eta)}. \quad (38)$$

The substitution of Eq. (29) into Eq. (28), yields

$$K_{III-V} = \frac{V}{R^{1.5}} F(\eta), \quad (39)$$

with

$$F(\eta) = \frac{4}{3} \frac{s}{R} \left(\frac{h}{R} \right)^{1.5} \left(\frac{1}{2} - \eta \right) \sqrt{2\pi\beta}. \quad (40)$$

The geometry function $F(\eta)$ related to the shear stress τ_{zx} is plotted in Fig. 5, *c* vs. the dimensionless ratio s/h_1 for varying lengths of the precrack a/D . A comparative assessment of the proposed formulation is performed with respect to previsions by Tada [8], adapted for a circular section, given by

$$F(\eta)_{\text{Tada}} = \frac{1}{\pi} \frac{s}{R} \sqrt{\frac{h}{R}} \pi \eta \sqrt{\frac{\pi \eta}{\sin \pi \eta}}, \quad (41)$$

where a quite good agreement is reached between the two different formulations, especially for lower values of s/h_1 and a/D .

From Fig. 5, *c*, it can be noticed, once again, that the increasing values of a/D always yield to an increasing geometry function $F(\eta)$ whose value increases from zero up to a maximum value, for s/h_1 ranging between zero and 0.7, thus decreasing for higher values of s/h_1 up to the unit value.

Torsional Moment. Consider now the circular cross section in Fig. 6, *a* subjected to a torsional moment T . In this case, the torsional shear stress τ'_{zx} parallel to the crack tip is related to the mode-III fracturing as follows

$$\tau'_{zx} = \frac{T}{J_P} \rho \cos \varphi = \frac{T}{J_P} y, \quad (42)$$

where J_P is the polar moment of inertia of the circular section.

If we consider that the torsional stress distribution in the reduced cross section is equal to the corresponding distribution in the whole circular section, except for the fact that the reference axis is shifted by a distance e from the original coordinate, the stress distribution on the reduced cross section passing through the crack tip can be expressed as

$$\tau'_{zx-c} = \frac{T}{J_c} y', \quad (43)$$

where J_c is the polar moment of inertia for the reduced portion of the circular cross section defined in dimensionless form as

$$\frac{J_c}{R^4} = \frac{\pi}{2} - \frac{1}{2} \arccos \left(1 - \frac{2a}{D} \right) + \frac{(1-2a/D)(3-8a/D+8(a/D)^2) \sqrt{a/D-(a/D)^2}}{3}. \quad (44)$$

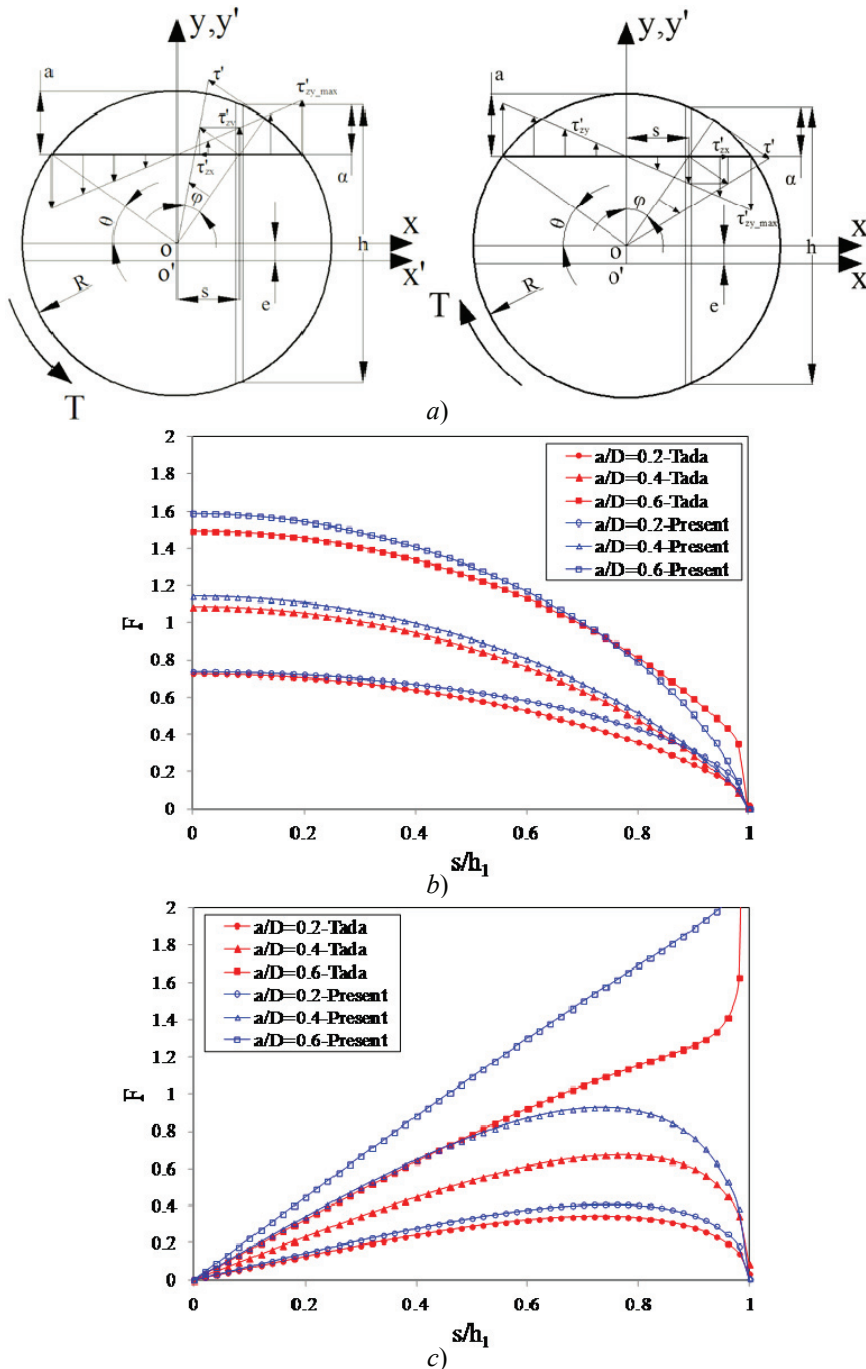


Fig. 6. Geometry function for K_{III-T} (a) and K_{II-T} (b) due to a torsional moment vs. the dimensionless ratio s/h_1

The singular stress component related to the mode-III SIF is

$$\tau'_{zx-s} = \frac{K_{III-T}}{\sqrt{2\pi r}} \quad (45)$$

The equilibrium of the forces along the x -direction of the beam is expressed as

$$\int_{\frac{h}{2}-a-b}^{\frac{h}{2}} \tau'_{zx} dy = \int_0^b \tau'_{zx-s} dr, \quad (46)$$

where b can be determined by enforcing the following condition

$$\tau'_{zx-s}|_{r=b} = \tau'_{zx-c}|_{y'=y=-\frac{h}{2}-a+e} \quad (47)$$

This yields

$$K_{III-T} = \frac{T}{R^{2.5}} F(\eta), \quad (48)$$

where

$$F(\eta) = \frac{1}{2} \sqrt{2\pi\beta} \left(\frac{R^4}{J_c} \right) \left(\frac{h}{R} \right)^{1.5} (1-\eta) \quad (49)$$

and

$$\beta = \frac{1}{2} - \eta - \frac{\pi}{2} \left(\frac{R^4}{J_c} \right) (1-\eta) + \sqrt{\frac{1}{4} \left(2\eta - 1 + \pi \left(\frac{R^4}{J_c} \right) (1-\eta) \right)^2 + \eta(1-\eta)}. \quad (50)$$

A comparative evaluation of the results in terms of $F(\eta)$ as given by Eq. (49) and from the literature by Tada [8] is given in Fig. 6, b. Figure 6, b shows a comparative evaluation of the proposed formulation with respect to solution proposed by Tada [8], and adapted for the a circular section as follows

$$F(\eta)_{Tada} = \frac{h}{\pi R} \sqrt{\frac{h}{R}} \pi \eta \sqrt{\frac{2}{\pi \eta} \tan\left(\frac{\pi}{2} \eta\right)}, \quad (51)$$

where a quite good agreement is reached between the two different formulations.

The vertical shear stress induced by the torsional moment is related to mode II, and it is expressed as follows

$$\tau'_{zy} = \frac{T}{J_P} \rho \sin\varphi = \frac{T}{J_P} x, \quad (52)$$

whose value is independent of the crack depth a , whereas it reaches a minimum value $\tau'_{zy-min} = 0$ in the middle of the crack tip $x = 0$ and a maximum value $\tau'_{zy-max} = \frac{T}{J_P} \sqrt{R^2 - y^2}$ at $x = -R \cos\theta = -\sqrt{R^2 - y^2}$.

Thus, the tensional stress of the beam under the torsional moment is based on the double definition as in the following

$$\tau'_{zy-c} = \frac{T}{J_c} s \quad (53)$$

and its singular form

$$\tau'_{zy-s} = \frac{K_{II-T}}{\sqrt{2\pi r}}. \quad (54)$$

By equating the expressions (53) and (54) for $y' = \bar{y} = \frac{h}{2} - \alpha + e$ we obtain the following expression for K_{II-T}

$$K_{II-T} = \frac{T}{J_c} s \sqrt{2\pi b}, \quad (55)$$

where b can be determined with the section method by considering the following relation

$$\int_{\frac{h}{2}-\alpha-b}^{h/2} \tau'_{zy} dy = \int_0^b \tau'_{zy-s} dr. \quad (56)$$

Thus, based on the Eqs. (55) and (56), b is computed in dimensionless form as

$$\beta = \eta \frac{J_c}{2J_p - J_c}, \quad (57)$$

while the substitution of Eq. (57) in the Eq. (55) yields to the following expression for the mode-II SIF

$$K_{II-T} = \frac{T}{R^{2.5}} F(\eta), \quad (58)$$

with

$$F(\eta) = \frac{R^4}{J_c} \frac{s}{R} \sqrt{2\pi\beta} \frac{h}{R}. \quad (59)$$

Figure 6, c plots the main results in terms of $F(\eta)$ obtained with our proposed formulation, compared to the corresponding predictions from the literature [8], adapted for the a circular section as in the following

$$F(\eta)_{\text{Tada}} = \frac{2s}{\pi R} \sqrt{\frac{h}{R}} \pi \eta \frac{1.122 - 0.56\eta + 0.085\eta^2 + 0.18\eta^3}{\sqrt{1-\eta}}. \quad (60)$$

A good agreement is reached between the two different formulations only for lower values of a/D , with an increasing discrepancy for higher ratios of a/D and s/h_1 . This is, mainly, related to the 2D approximated formulation provided by Tada [8] instead of the more appropriate 3D theory herein proposed for the purpose. For a/D ratios lower than 0.5, the geometry function $F(\eta)$ seems to increase from zero up to a maximum value for s/h_1 ranging between zero and 0.7, thus decreasing up to the null value for s/h_1 approaching the unit value. Differently, $F(\eta)$ seems to increase monotonically for a/D ratios higher than 0.5, in accordance to the predictions by Tada [8].

Comparison with Numerical Results Based on XFEM

Basis Notions of XFEM and Computation of the Mixed-Mode SIF. A short description of the SIF in mixed-mode conditions is provided in this section, remembering that the complete SIF for a 3D structure can be easily obtained as an algebraic summation of the components K_I , K_{II} and K_{III} defined as:

$$\begin{aligned} K_I &= \frac{M}{R^{2.5}} F_I(\eta); \\ K_{II} &= K_{II-V} + K_{II-T} = \frac{V}{R^{1.5}} F_{II}(\eta); \\ K_{III} &= K_{III-V} + K_{III-T} = \frac{T}{R^{2.5}} F_{III}(\eta). \end{aligned} \quad (61)$$

All the geometric functions determined analytically are now compared with those ones proposed in the literature [8] or obtained numerically with the commercial software ABAQUS by applying the XFEM approach. For the sake of brevity, the detailed implementation of the XFEM is not reported since it can be found in some specific references within the ABAQUS code.

The ABAQUS software provides the following methods to study fracture mechanics problems. In other words, quasi-statics can be applied to study the crack onset, growth and propagation along predefined paths, including the low-cycle fatigue, through the adoption of contour integrals. Part-through cracks in shells can be modeled inexpensively by using some line spring elements within a static procedure. An XFEM-based procedure models a crack as an enriched feature by adding degrees of freedom in elements with special displacement functions. The XFEM can be applied to simulate initiation and propagation of a discrete crack along an arbitrary, solution-dependent path without any requirement of remeshing. In addition, XFEM can be adopted to evaluate the contour integral without the need of refining the mesh around the crack tip (see Fig. 7). This approach also removes the requirement

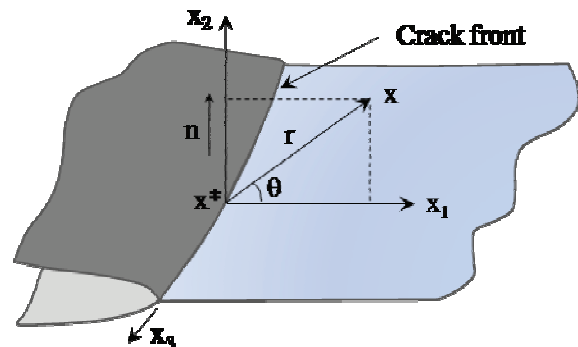


Fig. 7. Coordinate configuration for the crack front enrichment functions

of defining explicitly the crack front or specifying the virtual crack extension direction when evaluating the contour integral. The data required for the contour integral are determined automatically based on the level set signed distance functions at nodes within an element.

As mentioned before, SIFs K_I , K_{II} and K_{III} play a key role in LEFM, since they characterize the influence of load or deformation on the magnitude of the crack-tip stress and strain fields and measure the entity of crack propagation or crack driving forces. Furthermore, the stress intensity can be related to the energy release rate, i.e. the J -integral, computed directly in ABAQUS for a linear elastic material, as follows

$$J = \frac{1}{E} (K_I^2 + K_{II}^2) + \frac{1}{2\mu} K_{III}^2, \quad (62)$$

with $\bar{E} = E$ for plane stress and $\bar{E} = E/(1-\nu^2)$ for plane strain conditions. However, for a crack under a mixed-mode loading, the integral method [82] is implemented in the ABAQUS code to compute all the SIFs. This method is well known to be applicable to

cracks in isotropic and anisotropic linear materials. Figures 8, *a – i* show the XFEM-based results in terms of geometry functions $F_I(\eta)$, $F_{II}(\eta)$ and $F_{III}(\eta)$, for a single beam and the whole structure, compared to the theoretical predictions provided by our formulation and proposed in the literature by Tada [8]. All these plots clearly show the quite good agreement between results, especially for small values of s/h_1 . Increasing deviations between theoretical and numerical results can be observed for larger values of s/h_1 together with some visible discontinuities in the X-FEM trend for s/h_1 ratios approaching the unit value, where the solution could be affected by some boundary effects.

Computation of the Crack Propagation. For completeness of the analysis, we finally apply the main values of the SIFs as given by our proposed model to determine the crack propagation angle θ_0 (see Fig. 9) both numerically and analytically. In the last case, we apply three different fracture criteria, i.e. the criterion by Richard [64] (here labeled as CR1), the minimum strain energy density factor criterion by Sih [62, 63] (here labeled as CR2), and the criterion by Schollmann et al. [72] (here labeled as CR3). The analytical results

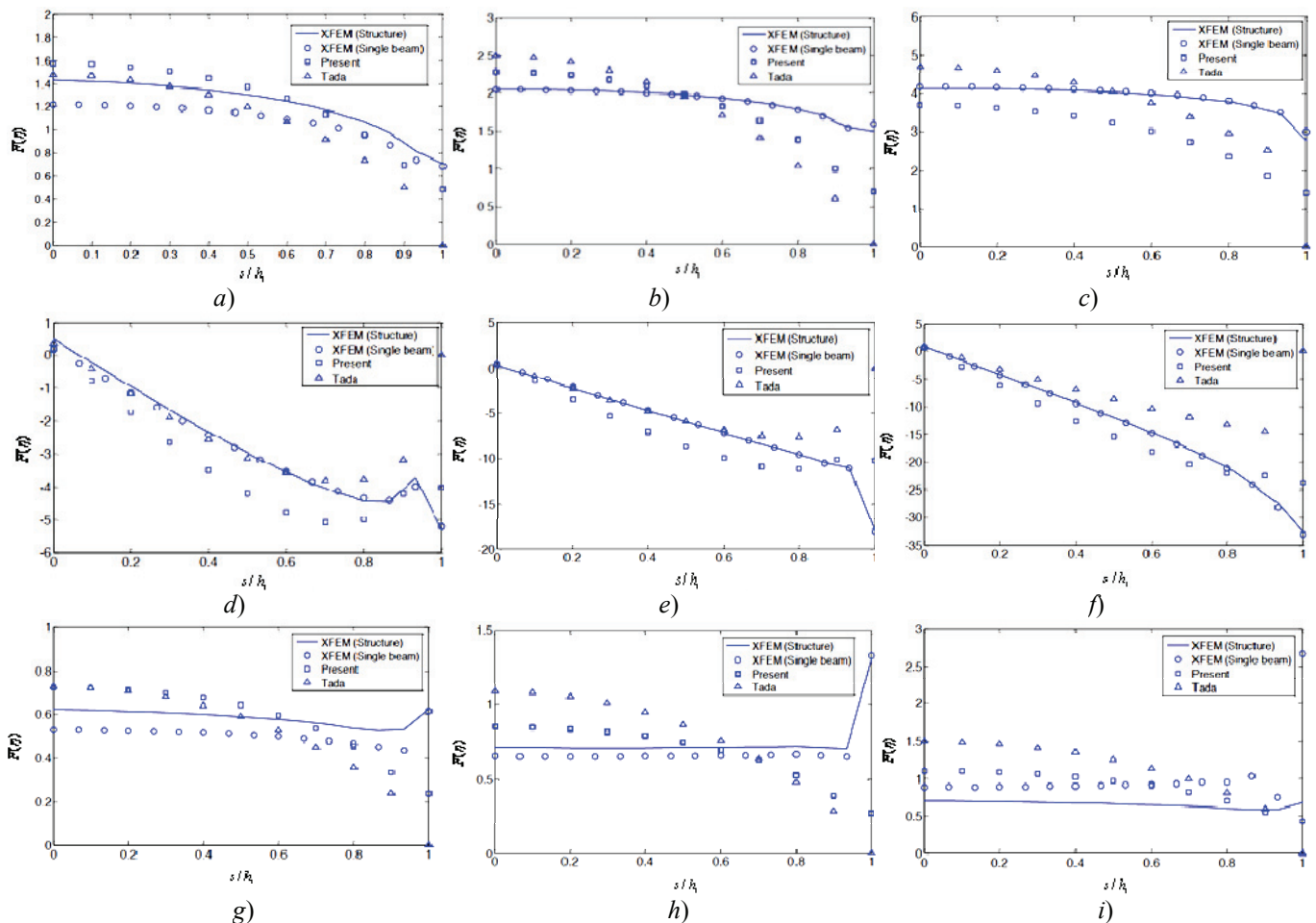


Fig. 8. Geometry functions for K_I : $a/D = 0.2$ (a), $a/D = 0.4$ (b), $a/D = 0.6$ (c); K_{II} : $a/D = 0.2$ (d), $a/D = 0.4$ (e), $a/D = 0.6$ (f); K_{III} : $a/D = 0.2$ (g); $a/D = 0.4$ (h); $a/D = 0.6$ (i), vs. the dimensionless ratio s/h_1

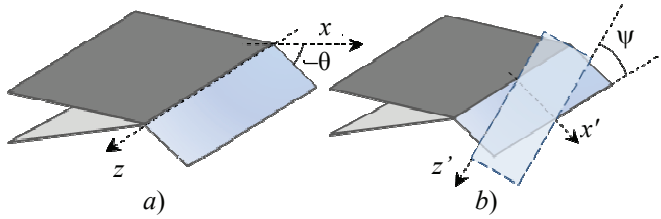


Fig. 9. Schematic representation of the negative tilting angle θ (a) and twisting angle ψ (b)

are compared with the corresponding predictions provided by XFEM.

Additional details about the three selected criteria are given in Appendix A, B and C, respectively. Figures 10, a – c show the tilt angle θ_0 as given by the

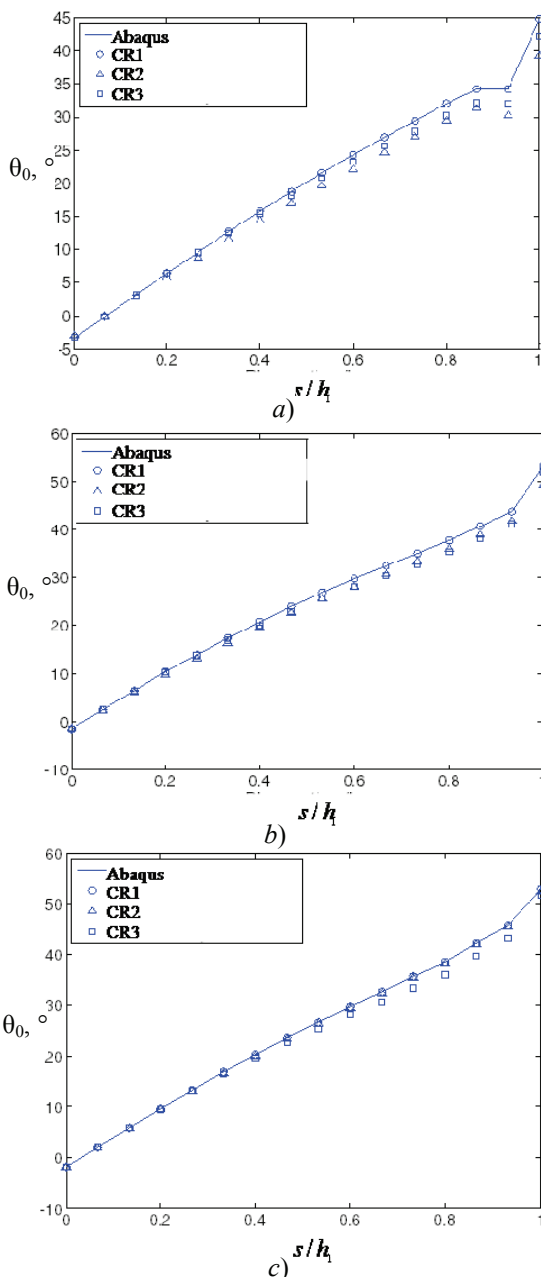


Fig. 10. Propagation angle θ_0 vs. the dimensionless ratio s/h_1 for $a/D = 0.2$ (a), $a/D = 0.4$ (b), $a/D = 0.6$ (c)

three cracking criteria, which clearly show a very good agreement of the results with those ones obtained numerically with the ABAQUS code. This confirms once more the ability of our proposed theoretical formulation combined with the selected criteria of fracturing to capture well the cracking development under mixed mode loading conditions. As also seen from Figs. 10, a – c, the propagation angle increases for increasing values of the dimensionless ratio s/h_1 and it seems to be quite unaffected by the dimensionless crack depth a/D , especially for low values of the ratio s/h_1 .

Conclusion

An extended analytical model based on a section method has been presented in this work for a correct estimation of the SIFs and fracture direction of propagation within a cracked shaft under a coupled longitudinal, flexural and torsional loading conditions. The extended version of the section method here presented, considers the stress singularity at the tip of an elastic crack, and is based on an elementary beam theory equilibrium condition for internal forces evaluated in the cross-section throughout the crack tip. The quality of the analytical geometry functions in modes I, II, and III is evaluated by comparison with the theoretical predictions available from the handbooks. Based on a comparative assessment, and an error estimation of the geometry functions for K_I , K_{II} , and K_{III} , a reasonable quality of the results is demonstrated in most cases yielding to a percentage error within 20 %. The only exception is represented by the torsional case and the associated computation of K_{II_T} , whose quality is reached only for low values of the geometry cracking ratios a/D and s/h_1 . This is mainly related to the 2D approximated formulation provided in the handbooks [8] instead of the 3D analytical theory herein proposed for the purpose. A similar parametric investigation is also performed numerically via the XFEM-based approach as implemented in the ABAQUS code, for the computation of the geometry functions and propagation angle of cracking for 3D circular beams and structures. In the last case, the numerical results are compared to the analytical ones as given by three different fracture criteria selected from the literature, and here combined with the SIFs provided by our formulation. The cracking propagation angle under mixed mode loading condition is demonstrated to be accurately captured due to the perfect matching of the numerical and analytical curves. The mixed-mode geometry functions $F_I(\eta)$, $F_{II}(\eta)$, and $F_{III}(\eta)$, instead, are of acceptable accuracy especially for small values

of s/h_1 when the error maintains lower than 20 %. The present study is not applicable only to rectangular and circular shafts because it has been already shown in the literature that the applied methodologies can be employed for the evaluation of the SIFs in general shape beams such as T and I beam in [83]. It will be the scope of a future contribution to study general shape beams within the present theoretical framework. Such analyses can lead to a numerical modeling where the crack behavior is modeled through line-springs for approximating the structural behavior [84, 85].

Acknowledgments

The research topic is one of the subjects of the Center of Study and Research for the Identification of Materials and Structures (CIMEST) "M. Capurso" of the University of Bologna (Italy), supported by the National Natural Science Foundation Project of China (51474039, 51404046, U1361205), by the Scientific Research Foundation of State Key Laboratory of Coal Mine Disaster Dynamics and Control (2011DA105287-ZD201302, 2011DA105287-MS201403), by the Fundamental Research Funds for the Central Universities (106112015CDJXY240003) and by the Basic Research of Frontier and Application of Chongqing (cstc2015jcyjA90019).

Appendix A. The Criterion by Richard et al. [64]

According to the generalized fracture criterion by Richard et al. [64], an equivalent stress intensity factor K_{eq} is defined as follows

$$K_{eq} = \frac{K_I}{2} + \frac{1}{2} \sqrt{K_I^2 + 4(\alpha_1 K_{II})^2 + 4(\alpha_2 K_{III})^2} \leq K_{IC}. \quad (A1)$$

As a function of the SIFs K_I , K_{II} , and K_{III} , with $\alpha_1 = K_{IC}/K_{IIc} = 1.155$ and $\alpha_2 = K_{IC}/K_{IIIc} = 1.0$. Consequently, unstable crack growth occurs when K_{eq} exceeds the fracture toughness K_{IC} for mode I. For the determination of the crack deflection angle θ_0 there does exist a simple relation, which has been verified by a large number of experiments

$$\theta_0 = \mp \left[140 \frac{|K_{II}|}{K_I + |K_{II}| + |K_{III}|} - 70 \left(\frac{|K_{II}|}{K_I + |K_{II}| + |K_{III}|} \right)^2 \right], \quad (A2)$$

where $\theta_0 < 0^\circ$ for $K_{II} > 0$ and $\theta_0 > 0^\circ$ for $K_{II} < 0$ and $K_I \geq 0$.

The same approach could be also applied to compute the twisting angle ψ_0 as follows

$$\psi_0 = \mp \left[78 \frac{|K_{II}|}{K_I + |K_{II}| + |K_{III}|} - 33 \left(\frac{|K_{II}|}{K_I + |K_{II}| + |K_{III}|} \right)^2 \right], \quad (A3)$$

where $\psi_0 < 0^\circ$ for $K_{III} > 0$ and $\psi_0 > 0^\circ$ for $K_{III} < 0$ and $K_I \geq 0$.

Appendix B. The Strain Energy Density Factor Criterion by Sih [62, 63]

As far as brittle fracture is concerned, the strain energy density (S) theory, as formulated by Sih [62, 63] provides a more general treatment of fracture mechanics problems due to its ability to describe the multi-scale features of the material damage or the crack growth and propagation under mixed mode conditions. From the linear theory of isotropic and homogeneous elasticity, the factor S is found to feature a singularity of the order $1/r$ near the crack front as in the following

$$\frac{dW}{dV} = \frac{S}{r}, \quad (B1)$$

where W is the strain energy, and V is the volume. The core of the S – criterion is the parameter of strain energy density factor S which is a function of the SIFs for linear elasticity, defined as (see Figs. 9, a, b)

$$S(\theta, \psi) = \frac{1}{16\pi\mu \cos \psi} (a_{11}K_I^2 + 2a_{12}K_IK_{II} + a_{22}K_{II}^2 + a_{33}K_{III}^2). \quad (B2)$$

In the previous equation, a_{ij} (with $i, j = 1, 2, 3$) are some auxiliary functions which depend on the elastic properties (i.e. the Young modulus E , Poisson ratio ν , and shear modulus μ) of the material, as well as on the polar angle θ of the crack tip, as follows:

$$\begin{aligned} a_{11} &= (3 - 4\nu - \cos \theta)(1 + \cos \theta); \\ a_{12} &= 2 \sin \theta (\cos \theta - (1 - 2\nu)); \\ a_{22} &= 4(1 - \nu)(1 - \cos \theta) + (1 + \cos \theta)(3 \cos \theta - 1); \\ a_{33} &= 4. \end{aligned} \quad (B3)$$

Based on the assumption of the S – criterion, the prediction of the crack initiation and direction is based on the following relations:

$$\begin{aligned} \frac{\partial a_{11}}{\partial \theta} &= \frac{1}{16\pi\mu \cos \psi} 2 \sin \theta (\cos \theta + 2\nu - 1); \\ \frac{\partial a_{12}}{\partial \theta} &= \frac{1}{16\pi\mu \cos \psi} (2 \cos \theta (\cos \theta + 2\nu - 1) - 2 \sin^2 \theta); \\ \frac{\partial a_{22}}{\partial \theta} &= \frac{1}{16\pi\mu \cos \psi} 2 \sin \theta (1 - 2\nu - 3 \cos \theta) \end{aligned} \quad (B4)$$

and

$$\begin{aligned} \frac{\partial S}{\partial \theta} \Big|_{\theta=\theta_0} &= \sin \theta (\cos \theta - 0.4) K_I^2 + \\ &+ (\cos \theta (\cos \theta - 0.4) - \sin^2 \theta) 2 K_I K_{II} + \\ &+ \sin \theta (0.4 - 3 \cos \theta) K_{II}^2 = 0, \end{aligned} \quad (B5)$$

where ν is herein set to 0.3. Similarly, the crack angle $\psi = \psi_0$ is derived by minimizing S with respect to ψ , i.e. $\partial S / \partial \psi|_{\psi=\psi_0} = 0$.

Appendix C. The Maximum Principle Stress Criterion by Schollmann et al. [72]

The 3D criterion as proposed by Schollmann et al. [72], considers all the fracture modes I, II and III, and is otherwise called as maximum principle stress σ_1 -criterion, where σ_1 is the maximum principle stress on a virtual cylindrical surface around the crack front. The present criterion reduces to the well-known maximum tangential stress-criterion in case of in-plane mixed-mode loading conditions (i.e. modes I and II) along the crack front, and thus can be considered as its generalization in 3D. Based on this criterion, the crack kinks in mode II with an angle θ_0 (see Fig. 9, a) which is determined through the following equation (for more details, see [72])

$$-6K_I\Gamma - K_{II}(6-12\Gamma^2) + \frac{(4K_I - 12K_{II}\Gamma)(-6K_I\Gamma - K_{II}(6-12\Gamma^2)) - 32K_{III}^2\Gamma(1+\Gamma^2)^2}{\sqrt{(4K_I - 12K_{II}\Gamma)^2 + 64K_{III}^2(1+\Gamma^2)^2}} = 0 \quad (C1)$$

where $\Gamma = \tan(\theta_0/2)$. At the same time, the crack front in mode III loading conditions grows into a new plane, which twists with an angle ψ_0 (see Fig. 9, b) determined as

$$\psi_0 = \frac{1}{2} \tan^{-1} \left(\frac{2K_{III}\Gamma}{K_I(\Gamma \cos^2(\theta_0/2) - 2\nu\Gamma) - K_{II}(3 + 3\sin^2(\theta_0/2) - 2\nu)} \right) \quad (C2)$$

An unstable crack growth occurs if the comparative SIF K_v reaches the fracture toughness K_{IC} , where K_v is expressed as

$$K_v = \frac{1}{2} \cos\left(\frac{\theta_0}{2}\right) \left\{ \int K_I \cos^2\left(\frac{\theta_0}{2}\right) - \frac{3}{2} K_{II} \sin(\theta_0) + \sqrt{\left[K_I \cos^2\left(\frac{\theta_0}{2}\right) - \frac{3}{2} K_{II} \sin(\theta_0) \right]^2 + 4K_{III}^2} \right\} \quad (C3)$$

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